

# A Mathematical Study of MHD Mixed Convection Channel Flow of Nanofluid

\*V. Ananthaswamy<sup>1</sup>, M. Subha<sup>2</sup> and S. Kaveri<sup>3</sup>

<sup>1</sup>Department of Mathematics, The Madura College, Madurai, Tamilnadu, India

<sup>2</sup>Department of Mathematics, MSNPM Women's College, Poovanthi, Tamilnadu, India

<sup>3</sup>M. Phil., Mathematics, The Madura College, Madurai, Tamilnadu, India

\*Corresponding author e-mail: ananthu9777@rediffmail.com

**Abstract:** The purpose of this article is to study and analyse the convective flow of nanofluid. The dimensionless entropy generation equation is obtained by solving the reduced momentum and energy equations. The momentum and energy equations are reduced to a system of ordinary differential equations by a similarity method. The Homotopy analysis method (HAM) is used to solve the resulting system of ordinary differential equations. The HAM is a valid mathematical tool for most of non-linear problems in science and engineering. The main purpose of the paper is to study the effects of Reynolds number, dimensionless temperature difference, Brinkman number, Hartmann number and other physical parameters on the entropy generation. These results are analysed and discussed.

**Keywords :** Channel flow; Mixed convection; Nanofluid; Entropy generation; Bejan number; Homotopy analysis method.

## 1. INTRODUCTION

The history of fluid flow is very old, one of the early studies is the work of Leonardo Da Vinci's which gave rapid advanced to the study of fluids mechanics about 500 years ago, but earlier than this time; prehistoric relics of irrigation canals have shown that the study of fluid behaviour were much more available by the time of ancient Egyptian (Nakayama and Boucher, 1999). Several centuries ago Johan and Daniel began more modern understanding of fluids motion known as Bernoulli's equation. Since then, many researchers have done numerous work on fluid mechanics.

The study of fluid flow and heat transfer in a vertical porous channel have been given considerable attention in the past few decades due to its wide applications in areas such as the design of cooling systems for electronic devices, chemical processing equipment, microelectronic cooling and solar energy (Jamalabadi et al., 2015). Numerous authors who conducted investigations on such flow include: Mutuku-Njaneand Makinde (2013) who investigated the combined effects of buoyancy force and Navier slip on MHD flow of a Nanofluid over a convectively heated vertical porous plate and submitted that increase in Grashof number decreases the fluid velocity. In the work of Jamalabadi et al. (2015), in which optimal design of Magnetohydrodynamic mixed convection flow in a vertical channel with slip boundary conditions and thermal radiation effects was analysed by using an entropy generation minimization method. It was concluded that Grashof numbers to Reynolds number ratio are better for maximizing the energy of the system. Adesanya and Falade (2015) presented the study on thermodynamic analysis for a third grade fluid through a vertical channel with internal heat generation, among the submissions was that increase in the Grashof number depletes the energy level of the thermal system. In addition, heat transfer dominates the channel with increase in Grashof number. Also, Sharma et al. (2014) investigated radiative

and free convective effects on MHD flow through a porous medium with periodic wall temperature and heat generation or absorption with the conclusion that increase in Grashof number increases the skin-friction coefficient at the wall.

## 2. Mathematical formulation of the problem

Consider the unsteady, incompressible Cu-water nanofluid in a vertical channel having permeable walls as shown in Fig. 1. The fluid is flowed at constant pressure gradient and an external uniform magnetic field is applied perpendicular to the plates. It is presumed that the fluid is injected uniformly into the channel at the fixed lower plate whereas the uniform fluid suction occurs at the moving upper plate. A transverse magnetic field with strength  $B_0$  is applied parallel to the Y-axis. The magnetic Reynolds number and the induced electric field are assumed to be small and negligible. The channel width is denoted by  $h$  with uniform temperature  $T_0$  at  $Y=0$  and temperature  $T_1$  at  $Y=h$  such that  $T_0 < T_1$ . The volume fraction of the Cu-nanoparticles in the base fluid is taken to be from 0 to 10% (i.e.  $\Phi = 0$  to 0.1) and assumed to have been mixed homogeneously with the base fluid under laboratory condition thus we have a single phase flow.

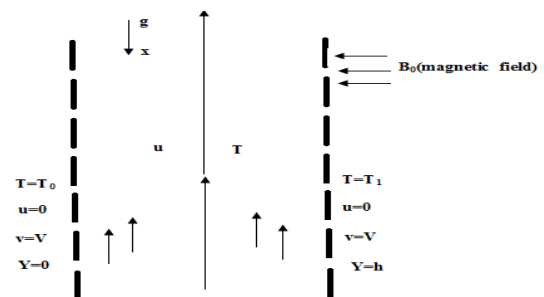


Fig.1:Schematic diagram of the problem

Using the Boussinesq approximation, the governing equations for the momentum and energy balance can be

expressed as [7,8,11,16,7,19];

$$-V \frac{du}{dy} + \beta_{nf} g(T - T_0) - \frac{1}{\rho_{nf}} \frac{dP}{dx} + \frac{\mu_{nf}}{\rho_{nf}} \frac{d^2u}{dy^2} - \frac{\sigma_{nf} B_0^2 u}{\rho_{nf}} = 0 \quad (1)$$

$$-(\rho C_P)_{nf} V \frac{dT}{dy} + k_{nf} \frac{d^2T}{dy^2} + \mu_{nf} \left( \frac{du}{dy} \right)^2 + \sigma_{nf} B_0^2 u^2 = 0 \quad (2)$$

$$E_g = \frac{k_{nf}}{T_0^2} \left( \frac{dT}{dy} \right)^2 + \frac{\mu_{nf}}{T_0} \left( \frac{du}{dy} \right)^2 + \frac{\sigma_{nf} B_0^2 u^2}{T_0} \quad (3)$$

With the following boundary conditions

$$\begin{aligned} u(0) = u(h) = 0 \\ T(0) = T_0, T(h) = T_1 \end{aligned} \quad (4)$$

where  $u$  is the axial velocity,  $P$  is the pressure,  $x$  is the axial distance,  $\rho_{nf}$  is the nanofluid density,  $\mu_{nf}$  is the nanofluid dynamic viscosity,  $(\rho C_P)_{nf}$  is the nanofluid specific heat capacity at constant pressure,  $k_{nf}$  is the nanofluid thermal conductivity,  $\sigma_{nf}$  is the nanofluid entropy

generation rate,  $T$  is the nanofluid temperature,  $\beta_{nf}$  is the nanofluid electrical conductivity,  $g$  is the gravitational acceleration and  $\sigma_{nf}$  is the nanofluid thermal expansion coefficient. In the eqns. (1), (2) and (3) the  $Cu$ -water nanofluid thermophysical properties are given by [17-21]

$$\begin{aligned} \rho_{nf} &= (1 - \Phi)\rho_{bf} + \Phi\rho_P, \quad (\rho C_P)_{nf} = (1 - \Phi)(\rho C_P)_{bf} + \Phi(\rho C_P)_P, \\ (\rho\beta)_{nf} &= (1 - \Phi)(\rho\beta)_{bf} + \Phi(\rho\beta)_P, \quad \frac{k_{nf}}{k_{bf}} = \frac{(k_P + 2k_{bf}) - 2\Phi(k_{bf} - k_P)}{(k_P + 2k_{bf}) + \Phi(k_{bf} - k_P)}, \quad (5) \\ \sigma_{nf} &= \sigma_{bf} \left[ 1 + \frac{3(\gamma - 1)\Phi}{(\gamma + 2) - (\gamma - 1)\Phi} \right], \quad \gamma = \frac{\sigma_P}{\sigma_{bf}}, \quad \mu_{nf} = \frac{\mu_{bf}}{(1 - \Phi)^{2.5}} \end{aligned}$$

**Table:1** Thermophysical properties of water and  $Cu$  nanoparticles [17-21]

| Materials   | $\rho(kg/m^3)$ | $C_p(J/kgK)$ | $k(W/mK)$ | $\sigma(S/m)$        | $\beta \times 10^5 (K^{-1})$ |
|-------------|----------------|--------------|-----------|----------------------|------------------------------|
| Pure water  | 997.1          | 4179         | 0.613     | $5.5 \times 10^{-6}$ | 21                           |
| Copper (cu) | 8933           | 385          | 401       | $59.6 \times 10^6$   | 1.67                         |

here  $\rho_{bf}$  is the base fluid density,  $\rho_P$  is the nanoparticles density,  $\mu_{bf}$  is the base fluid dynamic viscosity,  $k_{bf}$  is the basefluid thermal conductivity,  $k_P$  is the nanoparticles thermal conductivity,  $\beta_{bf}$  is the base fluid thermal

expansion coefficient,  $\beta_P$  is the nanoparticles thermal expansion coefficient,  $C_{Pbf}$  is the base fluid specific heat capacity,  $C_{PP}$  is the nanoparticles specific heat capacity,  $\sigma_{bf}$  is the base fluid electrical conductivity and  $\sigma_P$  is the nanoparticles electrical conductivity. Introducing

the following dimensionless parameters and variables,

$$\begin{aligned}
 W &= \frac{uh}{v_{bf}}, Gr = \frac{g\beta_{bf}(T_1 - T_0)h^3}{v_{bf}^2}, Pr = \frac{(\mu C_P)_{bf}}{k_{bf}}, Ha^2 = \frac{\sigma_{bf} B_0^2 h^2}{\mu_{bf}}, \\
 y &= \frac{Y}{h}, L = \frac{d\bar{P}}{dX}, Br = \frac{\mu_{bf} v_{bf}^2}{k_{bf}(T_1 - T_0)h^2}, A1 = Q \left[ 1 + \frac{3(\gamma - 1)\Phi}{(\gamma + 2) - (\gamma - 1)\Phi} \right], \\
 Q &= (1 - \Phi)^{2.5}, \theta = \frac{T - T_0}{T_1 - T_0}, M = \frac{(k_P + 2k_{bf}) + \Phi(k_{bf} - k_P)}{(k_P + 2k_{bf}) - 2\Phi(k_{bf} - k_P)}, \\
 \bar{P} &= \frac{Ph^2 \rho_{bf}}{\mu_{bf}^2}, Re = \frac{Vh}{v_{bf}}, Ec = \frac{v_{bf}^2}{C_{Pbf}(T_1 - T_0)h^2}, A3 = Q \left[ 1 - \Phi + \Phi \frac{(\rho\beta)_P}{(\rho\beta)_{bf}} \right], \\
 A4 &= \frac{M}{Q}, A5 = M \left[ 1 + \frac{3(\gamma - 1)\Phi}{(\gamma + 2) - (\gamma - 1)\Phi} \right], Ns = \frac{h^2 E_g T_0^2}{k_{bf}(T_1 - T_0)^2}, \\
 A2 &= Q \left[ 1 - \Phi + \Phi \frac{\rho_P}{\rho_{bf}} \right], A6 = M \left[ 1 - \Phi + \Phi \frac{(\rho C_P)_P}{(\rho C_P)_{bf}} \right], \\
 X &= \frac{x}{h}, v_{bf} = \frac{\mu_{bf}}{\rho_{bf}}, \gamma = \frac{\sigma_P}{\sigma_{bf}}
 \end{aligned} \tag{6}$$

For a given set of parameter values, the reactive convection flow evolves in time until a steady state condition is attained whenever this happens, then the equations becomes

$$\frac{d^2 w}{dy^2} = A2 Re \frac{dw}{dy} + A1 Ha^2 w - A3 Gr \theta - QL \tag{7}$$

$$\frac{d^2 \theta}{dy^2} = A6 Pr Re \frac{d\theta}{dy} - A4 Ec Pr \left( \frac{dw}{dy} \right)^2 - A5 Ec Pr Ha^2 w^2 \tag{8}$$

The corresponding boundary conditions are as follows:

$$\begin{aligned}
 w(0) &= 0, \theta(0) = 0 \\
 w(1) &= 0, \theta(1) = 1
 \end{aligned} \tag{9}$$

**Table: 2** Numerical values of the constants A1 to A6 can be determine as follows:

| <i>Constant</i><br>$\Phi$ | <i>A1</i> | <i>A2</i> | <i>A3</i> | <i>A4</i> | <i>A5</i> | <i>A6</i> | <i>Q</i> | <i>M</i> |
|---------------------------|-----------|-----------|-----------|-----------|-----------|-----------|----------|----------|
| 0                         | 1         | 1         | 1         | 1         | 1         | 1         | 1        | 1        |
| 0.05                      | 1.02      | 1.23      | 0.87      | 0.98      | 1         | 0.85      | 0.88     | 0.86     |
| 0.1                       | 1.02      | 1.38      | 0.75      | 0.97      | 1         | 0.74      | 0.77     | 0.75     |

By assumption, the exact solution of the eqn.(7) for the velocity of fluid is possible under this constant viscosity scenario and we obtain

$$w(y) = \frac{QLA1A2A3Gr}{2} (y - y^2) \quad (10)$$

In many engineering and industrial processes, entropy production destroys the available energy in the system. It is therefore imperative to determine the rate of entropy generation in a system, in order to optimize energy in the system for efficient operation in the system. The convection process in a channel is inherently irreversible and this causes continuous entropy generation. Using the eqn. (6), the dimensionless form of local entropy generation rate in an eqn. (3) is given as follows:

$$N_S = \frac{1}{M} \left( \frac{d\theta}{dy} \right)^2 + \frac{Br}{\Omega Q} \left[ \left( \frac{dw}{dy} \right)^2 + A1Ha^2w^2 \right] \quad (1)$$

where  $\Omega = \frac{T_w - T_0}{T_0}$  denotes the temperature

difference parameter. The Bejan number and the entropy generation rate are defined as follows:

$$Be = \frac{N_1}{N_S} = \frac{1}{1 + \phi} \quad (12)$$

and

$$N_S = N_1 + N_2 \quad (13)$$

where

$$N_1 = \frac{1}{M} \left( \frac{d\theta}{dy} \right)^2 \quad (14)$$

$$N_2 = \frac{Br}{\Omega Q} \left[ \left( \frac{dw}{dy} \right)^2 + A1Ha^2w^2 \right] \quad (15)$$

$$\lambda = \frac{N_2}{N_1} \quad (16)$$

Here  $N_1$  denotes heat transfer irreversibility due to heat transfer,  $N_2$  denote fluid friction and magnetic field irreversibility,  $\Phi$  denotes irreversibility ratio. The Bejan number as shown in an eqn.(12) has a range of  $0 \leq Be \leq 1$ . If  $Be = 0$ , then the irreversibility is dominated by the combined effects of fluid friction and magnetic fields, but if  $Be = 1$ , then the irreversibility due to heat transfer dominates the flow system by the virtue of finite temperature differences.

### 3 Solution of the non-linear boundary value problems using the Homotopy analysis method

In this paper an analytic tool for nonlinear problems, namely the Homotopy analysis method (HAM) by Liao [22], is employed as our basic concept to solve the nonlinear differential eqns., (3) and (4). The Homotopy analysis method is based on a music concept in topology, i.e. Homotopy by Hilton [23] which is widely applied in numerical techniques as in [24–27]. Unlike perturbation techniques [28-30], the Homotopy analysis method is independent of small/large parameters. Unlike all other reported perturbation and non-perturbation techniques such as the artificial small parameter method, the expansion method and Adomian decomposition method, the Homotopy analysis method provides us with a simple way to adjust and control the convergence region and rate of approximation series. The Homotopy analysis method has been successfully applied to many nonlinear problems such as viscous flows, heat transfer, nonlinear oscillations, nonlinear water waves, Thomas Fermi's atom model, etc.

In particular, by means of the Homotopy analysis method, the author Liao [33] gave a drag

formula for a sphere in a uniform stream, which agrees well with experimental results in a considerably larger region of Reynolds number than those of all reported analytic drag formulas. These successful applications of the Homotopy analysis method verify its validity for nonlinear problems in science and engineering. The

Homogony analysis method contains the auxiliary parameter  $h$ , which provides us with a simple way to adjust and control convergence region of solution series. The approximate analytical expressions of the velocity and temperature profiles using Homotopy analysis method are as follows:

$$w(y) = \frac{QLA1A2A3Gr}{2}(y - y^2) - h \left[ \begin{aligned} & \frac{QLA1A2^2 A3Gr Re}{2} \left( \frac{y^2}{2} - \frac{y^3}{3} \right) \\ & + \frac{QLA1^2 A2A3Gr Ha^2}{2} \left( \frac{y^3}{6} - \frac{y^4}{12} \right) \\ & - \left( \frac{-1}{2} \frac{y^2}{e^f - 1} + \frac{e^{f \cdot y}}{f^2(e^f - 1)} \right) A3Gr - \frac{QLy^2}{2} \\ & + t_1 y + t_2 \end{aligned} \right] \quad (17)$$

$$\theta(y) = C_1 + C_2 e^{f \cdot y} - h \left[ \begin{aligned} & \left( \frac{e^{f \cdot y}}{A4 A5 Ec(e^f - 1)} \right) \\ & - \left( \frac{A1^2 A2^2 A3^2 A4 Ec Pr Q^2 L^2 Gr^2}{192} \right) \cdot (1 - 2y^4) \\ & - \left( \frac{A1^2 A2^2 A3^2 A5 Ec Pr Q^2 L^2 Gr^2 Ha^2}{4} \right) \\ & \left( \frac{y^6}{30} - \frac{y^5}{10} + \frac{y^4}{12} \right) + t_3 y + t_4 \end{aligned} \right] \quad (18)$$

where

$$f = A4 A5 A6 Pr Re Ec \quad (19)$$

$$c_1 = \frac{-1}{e^f - 1} \quad (20)$$

$$c_2 = \frac{1}{e^f - 1} \quad (21)$$

$$t_1 = \left( \frac{-A_3 Gr}{f^2(e^f - 1)} \right) - \left( \frac{A_1^2 A_2 A_3 Gr Q L Ha^2}{24} \right) - \left( \frac{A_2^2 A_1 A_3 Gr Q L Re}{12} \right) + \left( \frac{-1}{2(e^f - 1)} + \frac{e^f}{f^2(e^f - 1)} \right) A_3 Gr + \left( \frac{Ql}{2} \right) \quad (22)$$

$$t_2 = \frac{A_3 Gr}{f^2(e^f - 1)} \quad (23)$$

$$t_3 = \left[ \frac{-e^f}{f^2(e^f - 1)} \right] + \left( \frac{A_1^2 A_2^2 A_3^2 Gr^2 Q^2 L^2 Ha^2 Ec Pr A_5}{240} \right) + \frac{1}{A_4 A_5 Ec (e^f - 1)} \quad (24)$$

$$t_4 = \left( \frac{A_1^2 A_2^2 A_3^2 Gr^2 Q^2 L^2 Ec Pr A_4}{192} \right) - \left( \frac{1}{A_4 A_5 Ec (e^f - 1)} \right) \quad (25)$$

#### 4. Results and discussion

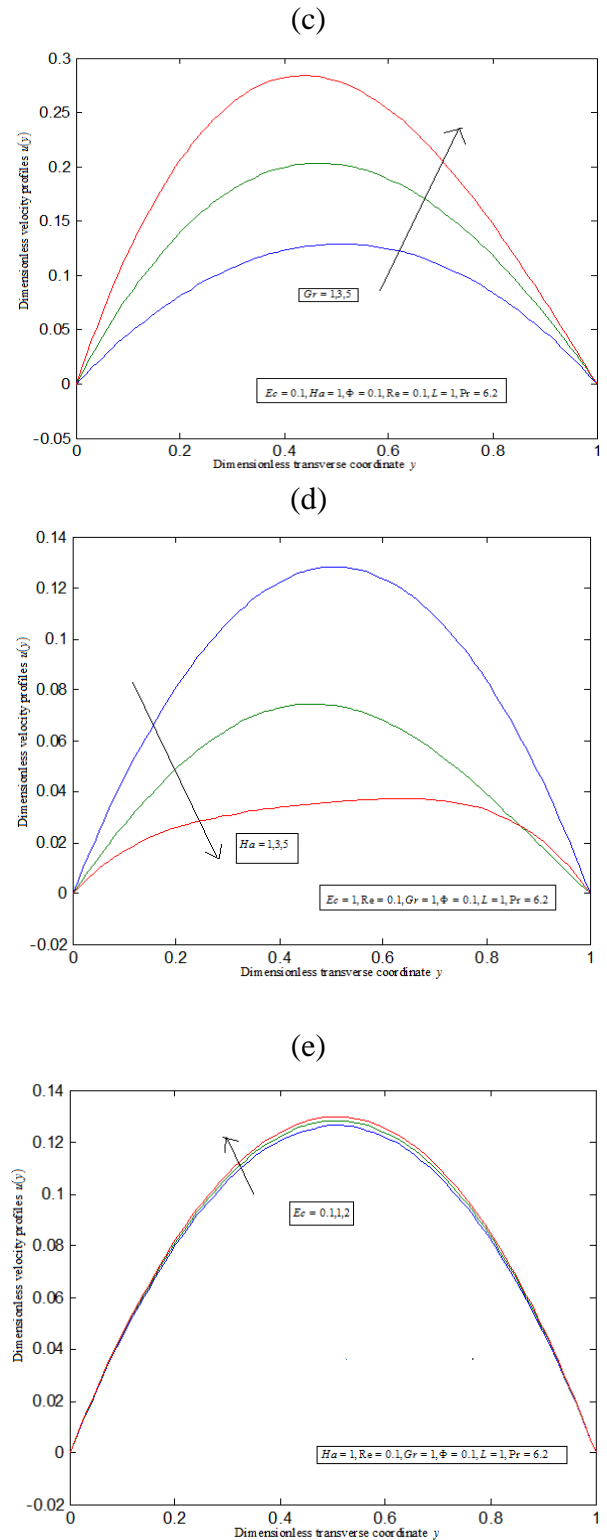
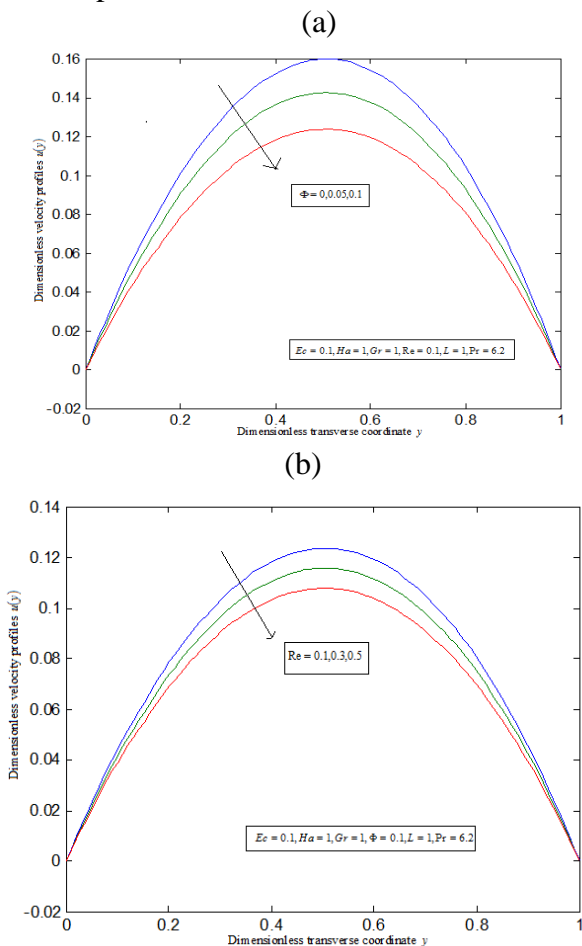
Figure 1 shows the schematic diagram of the problem. Fig.2 (a)-(e) represents the dimensionless transverse coordinate versus dimensionless fluid velocity. From Fig. 2 (a), it is clear that when irreversibility ratio increases the corresponding dimensionless fluid velocity profiles decreases in some fixed values of the other parameters  $L, Pr, Re, Ec, Ha, Gr$  and  $h$ . From Fig. 2 (b), it is observed that when the Reynolds number increases the corresponding dimensionless fluid velocity profiles decreases in some fixed values of the other parameters. From Fig.2 (c), it is inferred that when the Grashof number increases the corresponding dimensionless fluid velocity profiles increases in some fixed values of the other parameters. From Fig. 2 (d), it is depict that when the Hartmann number increases the corresponding dimensionless fluid velocity profiles decreases in some fixed values of the other parameters. From Fig. 2 (e), it is noted that when the Eckert number increases the corresponding dimensionless fluid velocity profiles increases in some fixed values of the other parameters.

Figure 3(a)-(d) are representing the dimensionless transverse coordinate versus Dimensionless fluid temperature profiles. From Fig.3 (a), it is inferred that the irreversibility ratio increases fluid temperature profiles increases in some fixed values of the other parameters  $L, Pr, Re, Ec, Ha, Gr$  and  $h$ . From Fig. 3 (b), it is observed that when the Reynolds number increases the corresponding dimensionless fluid temperature profiles decreases in some fixed values of the other parameters. From Fig. 3 (c), it is clear that when the Grashof number increases the corresponding dimensionless fluid temperature profiles increases in some fixed values of the other parameters. From Fig. 3 (d), it is depict that when the Hartmann number increases the corresponding dimensionless fluid temperature profiles decreases in some fixed values of the other parameters.

Figure 4 (a)-(c) are represent the Entropy generation rate versus dimensionless transverse coordinate. From Fig.4 (a) it is inferred that the irreversibility ratio increases entropy generation rate increases in some fixed values of the other parameters  $L, Pr, Re, Ec, Ha, Gr, Br\Omega^{-1}$  and  $h$ .

From Fig. 4 (b), it is observed that when the Reynolds number increases the corresponding entropy generation rate decreases in some fixed values of the other parameters. From Fig. 4 (c), it is noted that when the  $Br\Omega^{-1}$  increases the corresponding entropy generation rate increases in some fixed values of the other parameters.

Figure 5 (a)-(c) are represent the dimensionless transverse coordinate versus Bejan number. From Fig.5 (a) it is inferred that the irreversibility ratio increases Bejan number increases in some fixed values of the other parameters  $L, Pr, Re, Ec, Ha, Gr, Br\Omega^{-1}$  and  $h$ . From Fig. 5 (b), it is observed that when the Reynolds number increases the corresponding Bejan number decreases in some fixed values of the other parameters. From Fig. 5 (c), it is noted that when the  $Br\Omega^{-1}$  increases the corresponding Bejan number decreases in some fixed values of the other parameters.





**Fig. 2 (a)-(e):** Dimensionless fluid velocity profiles  $w(y)$  versus dimensionless transverse coordinate  $(y)$ . The curves are plotted using the eqn. (17) for various values of the dimensionless parameter  $Pr, L$  and in some fixed values of the other dimensionless parameters, when

(a)  $Ec, Ha, Gr, Re, \Phi = 0, h = -0.20, Ec, Ha, Gr, Re, \Phi = 0.05, h = -0.15,$

$Ec, Ha, Gr, Re, \Phi = 0.1, h = -0.16$

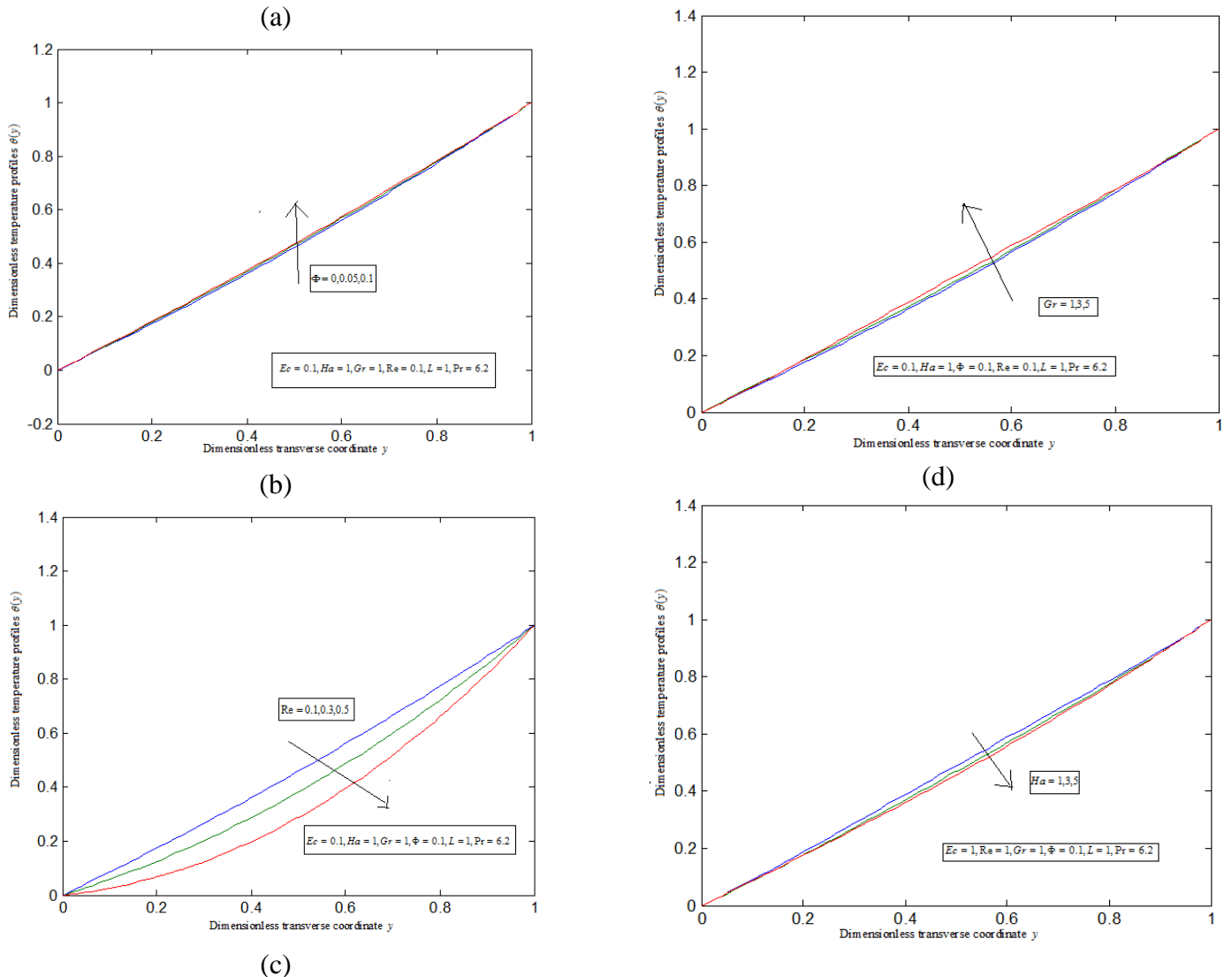
(b)  $Ec, Ha, Gr, Re = 0.1, \Phi, h = -0.16, Ec, Ha, Gr, Re = 0.3, \Phi, h = -0.10,$

$Ec, Ha, Gr, Re = 0.5, \Phi, h = -0.04$

(c)  $Ec, Ha, Gr = 1, Re, \Phi, h = -0.20, Ec, Ha, Gr = 3, Re, \Phi, h = 0.52, Ec, Ha, Gr = 5, Re, \Phi, h = 0.855$

(d)  $Ec, Ha = 1, Gr, Re, \Phi, h = -0.20, Ec, Ha = 3, Gr, Re, \Phi, h = 0.72, Ec, Ha = 5, Gr, Re, \Phi, h = -0.49$

(e)  $Ec = 0.1, Ha, Gr, Re, \Phi, h = -0.18, Ec = 1, Ha, Gr, Re, \Phi, h = -0.20, Ec = 2, Ha, Gr, Re, \Phi, h = -0.22$



**Fig. 3 (a)-(d) :** Dimensionless fluid temperature profiles  $\theta(y)$  versus dimensionless transverse coordinate  $(y)$ . The curves are plotted using the eqn. (18) for various values of the dimensionless parameter  $Pr, L$  and in some fixed values of the other dimensionless parameters, when



(a)  $Ec, Ha, Gr, Re, \Phi = 0, h = -0.45, Ec, Ha, Gr, Re, \Phi = 0.05, h = -0.43,$

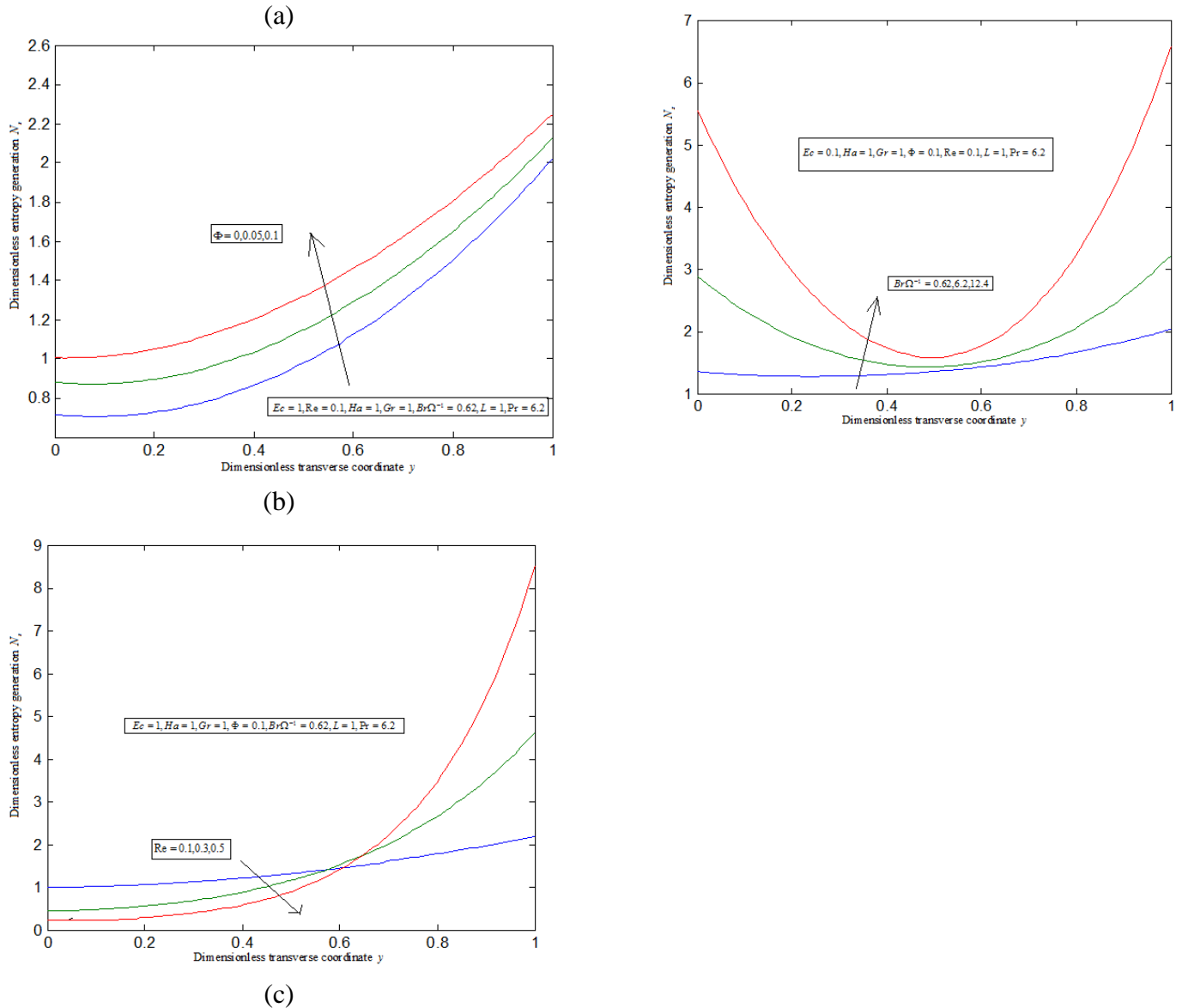
$Ec, Ha, Gr, Re, \Phi = 0.1, h = -0.40$

(b)  $Ec, Ha, Gr, Re = 0.1, \Phi, h = -0.65, Ec, Ha, Gr, Re = 0.3, \Phi, h = -0.60,$

$Ec, Ha, Gr, Re = 0.5, \Phi, h = -0.65$

(c)  $Ec, Ha, Gr = 1, Re, \Phi, h = -0.59, Ec, Ha, Gr = 3, Re, \Phi, h = -0.72, Ec, Ha, Gr = 5, Re, \Phi, h = 0.55$

(d)  $Ec, Ha = 1, Gr, Re, \Phi, h = 0.80, Ec, Ha = 3, Gr, Re, \Phi, h = -0.7, Ec, Ha = 5, Gr, Re, \Phi, h = 0.01$



**Fig. 4 (a)-(c) :** Dimensionless entropy generation rate  $N_s$  versus dimensionless transverse coordinate ( $y$ ). The curves are plotted using the eqn. (11) for various values of the dimensionless parameter  $Pr, L$  and in some fixed values of the other dimensionless parameters, when

(a)  $Ec, Ha, Gr, Re, Br\Omega^{-1}, \Phi = 0, h = -0.10, Ec, Ha, Gr, Br\Omega^{-1}, Re, \Phi = 0.05, h = -0.10,$

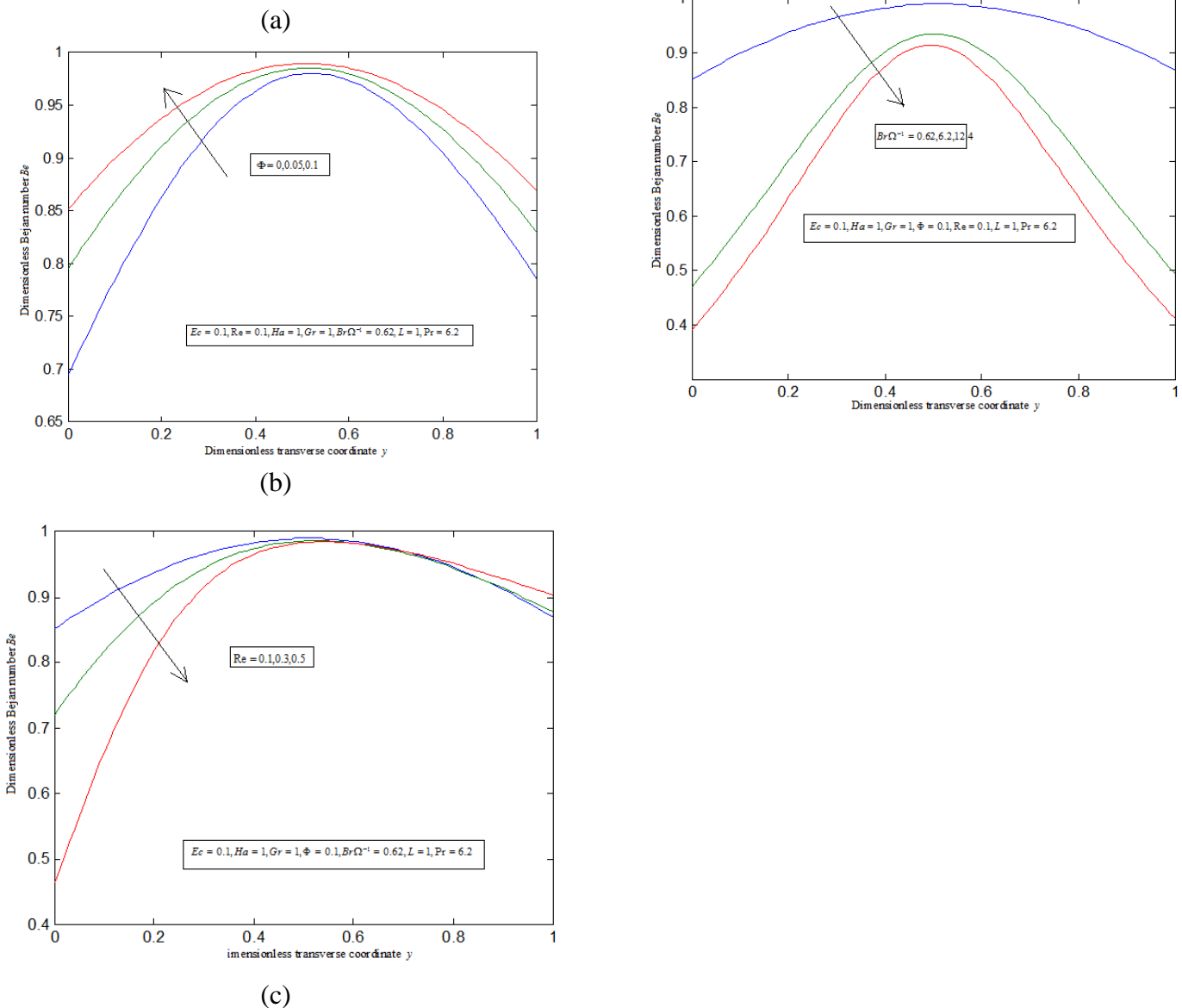
$Ec, Ha, Gr, Re, Br\Omega^{-1}, \Phi = 0.1, h = -0.11$

(b)  $Ec, Ha, Gr, Br\Omega^{-1}, Re = 0.1, \Phi, h = -0.03, Ec, Ha, Gr, Br\Omega^{-1}, Re = 0.3, \Phi, h = -0.03,$

$Ec, Ha, Gr, Br\Omega^{-1}, Re = 0.5, \Phi, h = -0.01$

(c)  $Ec, Ha, Gr = 1, Br\Omega^{-1} = 0.62 Re, \Phi, h = -0.45, Ec, Ha, Gr = 3, Re, Br\Omega^{-1} = 6.2\Phi, h = -0.10,$

$Ec, Ha, Gr = 5, Re, \Phi, h = -0.25$



**Fig. 5 (a)-(c) :** Dimensionless Bejan number  $Be$  versus dimensionless transverse coordinate ( $y$ ). The curves are plotted using the eqn. (12) for various values of the dimensionless parameter  $Pr, L$  and in some fixed values of the other dimensionless parameters, when

$$(a) Ec, Ha, Gr, Re, Br\Omega^{-1}, \Phi = 0, h = -0.50, Ec, Ha, Gr, Br\Omega^{-1}, Re, \Phi = 0.05, h = -0.28,$$

$$Ec, Ha, Gr, Re, Br\Omega^{-1}, \Phi = 0.1, h = -0.20$$

$$(b) Ec, Ha, Gr, Br\Omega^{-1}, Re = 0.1, \Phi, h = -0.20, Ec, Ha, Gr, Br\Omega^{-1}, Re = 0.3, \Phi, h = -0.35,$$

$$Ec, Ha, Gr, Br\Omega^{-1}, Re = 0.5, \Phi, h = -0.43$$

$$(c) Ec, Ha, Gr = 1, Br\Omega^{-1} = 0.62 Re, \Phi, h = -0.20, Ec, Ha, Gr = 3, Re, Br\Omega^{-1} = 6.2\Phi, h = -0.02,$$

$$Ec, Ha, Gr = 5, Re, \Phi, h = -0.11$$

## 5. Conclusions

A mathematical study is carried out for the entropy generation rate in hydromagnetic mixed convection flow of Cu-Water nanofluid through a channel with permeable wall. The velocity and the temperature profiles are obtained graphically and analytically with the Homotopy analysis method. We can also derive the analytical expressions for the skin friction, Nusselt number, entropy generation number with the Bejan number. The analytical results were obtained through this paper demonstrates that the optimal design and the efficient performance of a flow system involving nanofluid or a thermally designed system can be improved by choosing the appropriate values of the physical parameters. The Homotopy analysis method (HAM) contains the convergence control parameter  $h$ , so that it can be extend to solve the other MHD fluid flow problems in other engineering and sciences.

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## Appendix A

### Approximate analytical expressions of the non-linear differential eqns. (7) and (8) using Homotopy analysis method

In this section, we indicate how the eqns. 17) and (18) are derived in this paper. To find the solution of the equation (3.7) and (3.8) and reduced to

$$\frac{d^2 w}{dy^2} - A2 \operatorname{Re} \frac{dw}{dy} - A1 Ha^2 w + A3 Gr \theta + QL = 0 \tag{B.1}$$

$$\frac{d^2 \theta}{dy^2} - A6 \operatorname{Pr} \operatorname{Re} \frac{d\theta}{dy} + A4 Ec \operatorname{Pr} \left( \frac{dw}{dy} \right)^2 + A5 Ec \operatorname{Pr} Ha^2 w^2 = 0 \tag{B.2}$$

We construct the HAM for (B.1) and (B.2) are as follows:

$$(1-p) \left( \frac{d^2 w}{dy^2} + QL \right) - hp \left( \frac{d^2 w}{dy^2} - A2 \operatorname{Re} \frac{dw}{dy} - A1 Ha^2 w + A3 Gr \theta + QL \right) = 0 \tag{B.3}$$

$$(1-p) \left( \frac{d^2 \theta}{dy^2} - A6 \operatorname{Pr} \operatorname{Re} \frac{d\theta}{dy} \right) - hp \left( \frac{d^2 \theta}{dy^2} - A6 \operatorname{Pr} \operatorname{Re} \frac{d\theta}{dy} + A4 Ec \operatorname{Pr} \left( \frac{dw}{dy} \right)^2 + A5 Ec \operatorname{Pr} Ha^2 w^2 \right) = 0 \tag{B.4}$$

The approximate solution of the equation (B.3) and (B.4) are as follows:

$$w = w_0 + pw_1 + p^2 w_2 + p^3 w_3 + p^4 w_4 + \dots \tag{B.5}$$

$$\theta = \theta_0 + p\theta_1 + p^2 \theta_2 + p^3 \theta_3 + p^4 \theta_4 + \dots \tag{B.6}$$

Substituting the eqns.(B.5) into an eqn. (B.6), we get the following results:

$$(1-p) \left( \frac{d^2 (w_0 + pw_1 + \dots)}{dy^2} + QL \right) - hp \left( \frac{d^2 (w_0 + pw_1 + \dots)}{dy^2} - A2 \operatorname{Re} \frac{d(w_0 + pw_1 + \dots)}{dy} - A1 Ha^2 (w_0 + pw_1 + \dots) + A3 Gr (\theta_0 + p\theta_1 + \dots) + QL \right) = 0 \tag{B.7}$$

$$(1-p) \left( \frac{d^2 (\theta_0 + p\theta_1 + \dots)}{dy^2} - A6 \operatorname{Pr} \operatorname{Re} \frac{d(\theta_0 + p\theta_1 + \dots)}{dy} \right) - hp \left( \frac{d^2 (\theta_0 + p\theta_1 + \dots)}{dy^2} - A6 \operatorname{Pr} \operatorname{Re} \frac{d(\theta_0 + p\theta_1 + \dots)}{dy} + A4 Ec \operatorname{Pr} \left( \frac{d(w_0 + pw_1 + \dots)}{dy} \right)^2 + A5 Ec \operatorname{Pr} Ha^2 (w_0 + pw_1 + \dots)^2 \right) = 0 \tag{B.8}$$

Comparing the coefficients of like powers of  $p^0, p^1$  in the eqns. (B.7) and (B.8) we get

$$p^0 : \frac{d^2 w_0}{dy^2} + QL = 0 \tag{B.9}$$

$$p^0 : \frac{d^2 \theta_0}{dy^2} - A6 \operatorname{Pr} \operatorname{Re} \frac{d\theta_0}{dy} = 0 \tag{B.10}$$

$$P^1 : \frac{d^2 w_1}{dy^2} - \frac{d^2 w_0}{dy^2} - QL - h \left( \frac{d^2 w_0}{dy^2} - A2 \operatorname{Re} \frac{dw_0}{dy} - A1 Ha^2 w_0 + A3 Gr \theta_0 + QL \right) = 0 \quad (B.11)$$

$$P^1 : \frac{d^2 \theta_1}{dy^2} - A6 \operatorname{Pr} \operatorname{Re} \frac{d\theta_1}{dy} - \frac{d^2 \theta_0}{dy^2} + A6 \operatorname{Pr} \operatorname{Re} \frac{d\theta_0}{dy} - h \left( \begin{array}{l} \frac{d^2 \theta_0}{dy^2} - A6 \operatorname{Pr} \operatorname{Re} \frac{d\theta_0}{dy} \\ + A4 Ec \operatorname{Pr} \left( \frac{dw_0}{dy} \right)^2 \\ + A5 Ec \operatorname{Pr} Ha^2 w_0^2 \end{array} \right) = 0 \quad (B.12)$$

The initial approximations are as follows:

$$w_0(0) = 0, \theta_0(0) = 0 \quad (B.13)$$

$$w_0(1) = 0, \theta_0(1) = 1$$

$$w_j(0) = 0, \theta_j(0) = 0$$

$$w_j(1) = 0, \theta_j(1) = 0, j = 1, 2, 3, \dots \quad (B.14)$$

Solving the eqns. (B.9) - (B.12) and using the initial approximations (B.13) and (B.14), we can obtain the following results:

$$w_0 = \frac{QLA1A2A3Gr}{2} (y - y^2) \quad (B.15)$$

$$w_1 = -h \left[ \begin{array}{l} \frac{QLA1A2^2 A3Gr \operatorname{Re} \left( \frac{y^2}{2} - \frac{y^3}{3} \right)}{2} \\ + \frac{QLA1^2 A2A3Gr Ha^2 \left( \frac{y^3}{6} - \frac{y^4}{12} \right)}{2} \\ - \left( \frac{-1}{2} \frac{y^2}{e^f - 1} + \frac{e^{f \cdot y}}{f^2 (e^f - 1)} \right) - \frac{QLy^2}{2} \\ + t_1 y + t_2 \end{array} \right] \quad (B.16)$$

$$\theta_0 = C_1 + C_2 e^{f \cdot y} \quad (B.17)$$

$$\theta_1 = -h \left[ \begin{array}{l} \frac{e^{f \cdot y}}{A4 A5 Ec (e^f - 1)} - \frac{A1^2 A2^2 A3^2 A4 Ec \operatorname{Pr} Q^2 L^2 Gr^2 (1 - 2y^4)}{192} \\ - \frac{A1^2 A2^2 A3^2 A5 Ec \operatorname{Pr} Q^2 L^2 Gr^2 Ha^2 \left( \frac{y^6}{30} - \frac{y^5}{10} + \frac{y^4}{12} \right)}{4} \\ + t_3 y + t_4 \end{array} \right] \quad (B.18)$$

where  $f, t_1, t_2, t_3, t_4$  are defined in the eqns.(7)-(21) respectively.

According to HAM, we can conclude that

$$w = \lim_{P \rightarrow 1} w(\eta) = w_0 + w_1 \quad (B.19)$$

$$\theta = \lim_{P \rightarrow 1} \theta(\eta) = \theta_0 + \theta_1 \quad (B.20)$$

After putting the eqns. (B.15) and (B.16) into an eqn.(B.19) and putting the eqns.(B.17) and (B.18) into an eqns.(B.20), we can obtain the solution in text eqns. (17) and (18).

### Appendix: B – Nomenclature

| Symbol   | Meaning                             |
|----------|-------------------------------------|
| $u, v$   | Velocities of the fluid             |
| $\rho$   | Fluid density                       |
| $p$      | Fluid pressure                      |
| $x, y$   | Coordinates                         |
| $L$      | axial pressure gradient parameter   |
| $T_0$    | Temperature at the lower plate      |
| $T_1$    | Temperature at upper plate          |
| $T$      | Fluid temperature                   |
| $\mu$    | Coefficient of viscosity            |
| Pr       | Prandtl number                      |
| $Ha$     | Hartmann number                     |
| $Gr$     | Grashof number                      |
| $Be$     | Bejan number                        |
| $N_s$    | Entropy generation rate             |
| $\theta$ | Dimensionless temperature           |
| Re       | Reynolds number                     |
| $Ec$     | Eckert number                       |
| $Pe$     | Peclet number                       |
| $Br$     | Brinkmann number                    |
| $\Omega$ | Temperature difference parameter    |
| $y$      | Dimensionless transverse coordinate |
| $\phi$   | Irreversibility ratio               |